

1 Thermal calibration of the MEDA-TIRS radiometer onboard NASA's Perseverance rover

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11 Abstract

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14 This article describes a comprehensive testing method for the thermal calibration of the Thermal
15 InfraRed Sensor (TIRS) onboard NASA's *Perseverance* rover. Ground-based IR detectors
16 operating at the surface of Mars are subjected to inaccuracies caused by the inclusion of thermal
17 gradients in their packages. To reduce such uncertainties, it is necessary to compensate for their
18 effects. Here, details of the TIRS thermo-mechanical design and a simplified thermal
19 mathematical model (TMM) that accounts for the presence of thermal gradients in the
20 detector's package are provided. Then, a set of equations for the estimation and compensation
21 of thermal gradients are proposed based on the results of the TMM. Thermal calibration tests
22 to identify the mathematical estimators are analyzed, providing details of the test setups and
23 highlighting their limitations and restrictions. Finally, experimental results of the calibration
24 tests are presented, along with the uncertainty sources and potential systematic errors
25 associated with the estimation of the gradients. The results presented here show a significant
26 improvement in the accuracy of TIRS versus previous work, thus fulfilling of the radiometer
27 scientific requirements set by the Mars 2020 science team.

28 Keywords

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30 Spacecraft IR radiometer, IR detector package gradients, Detector modeling, Thermal
31 calibration, Calibration accuracy

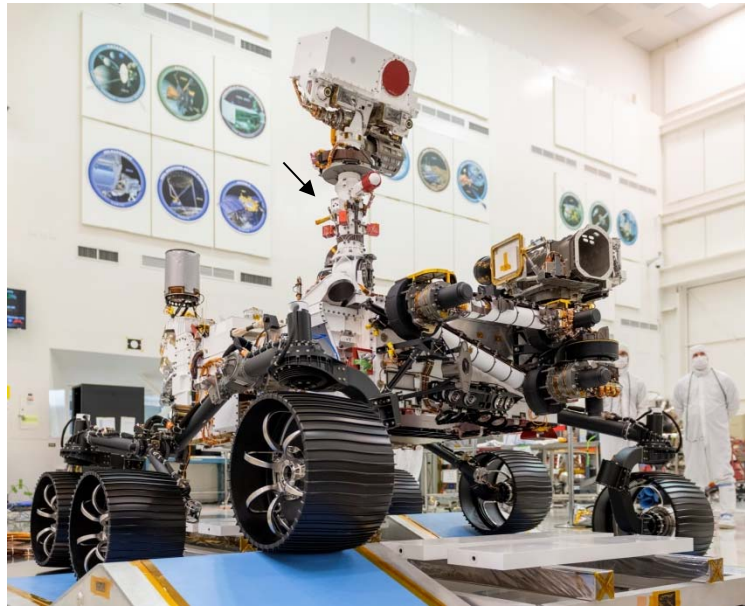
32 1. Introduction

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34 Among the different scientific instruments onboard NASA's Mars2020 *Perseverance* rover, the
35 suit of sensors making up the Mars Environmental Dynamic Analyzer (MEDA) instrument was
36 selected to characterize the environment in preparation for the human exploration of Mars,
37 which is one of the four objectives of the M2020 mission [1]. MEDA's specific objectives are to
38 characterize the atmospheric dust size and morphology to understand its effects on the
39 operation of surface systems and human health, and to validate global atmospheric models by
40 performing surface weather measurements. The in-situ meteorological measurements that will
41 be collected by MEDA (air and ground temperature, radiative fluxes, wind speed and direction,
42 pressure, and relative humidity) [2], together with the data collected by previous landing
43 missions [3, 4], will expand our current understanding of the near-surface environmental
44 conditions on Mars. These data are complementary to those obtained from Mars orbiters, which
45 cannot resolve the atmospheric layers closest to the surface where rovers and landers operate
46 [5]. A good characterization of the environment at surface level is also relevant to determine the
47 presence of certain compounds as well as the cycle of water and forms of life [4]. Despite the
48 vast amount of Mars surface data already available, there are still many strategic knowledge
49 gaps that can only be closed by sending new missions to our neighbor planet [5]. Specifically,

50 the MEDA *in-situ* space radiometer called the Thermal Infrared Sensor (TIRS) [6] (Fig. 1) is used
 51 for the validation of atmospheric models by measuring the net thermal infrared, solar reflected
 52 radiation and atmosphere and ground temperature at the surface, through different channels
 53 and measurement bands (Table 1).

Channel	Purpose	Range[μm]	Pointing angles
IR1	Downward LW flux	6.5-30	Upward (+35°)
IR2	Atmosphere temperature	14.5-15.5	Upward (+35°)
IR3	Reflected SW	0.3-3	Downward (-35°)
IR4	Upward LW flux	6.5-30	Downward (-35°)
IR5	Surface brightness temperature	8-14	Downward (-35°)

54 Table 1. TIRS channels.



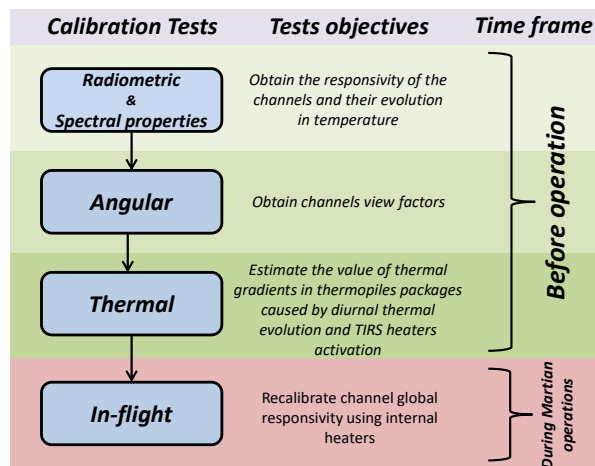
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 56 Figure 1. MEDA-TIRS located at NASA Perseverance rover remote sensing mast (black arrow,
 57 credit NASA-JPL).

58 One of the most important sources of uncertainty in infrared (IR) radiometry is the appearance
 59 of spatial thermal gradients in the package of the IR detectors [7, 8, 9, 10]. Therefore, a proper
 60 calibration of these gradients represents a key aspect to obtain reliable data. The main purpose
 61 of this article is to describe a comprehensive method for the compensation and calibration of the
 62 thermal gradients on the packages of the IR detectors. This calibration is part of the TIRS'
 63 calibration plan, whose general structure, test objectives and execution time frame are shown in
 64 (Fig. 2). Radiometric and angular calibration [11] tests have already been described in [12], while
 65 in-flight calibration tests will be discussed in forthcoming articles.

66 Section 2 of this article updates the description and justifies the final thermo-mechanical design
 67 of the TIRS flight model (FM), preliminarily presented for an engineering model in [6]. Additional
 68 information about isolation and fixing elements, optical treatments, and heater details is also
 69 provided. Section 3 includes a simplified mathematical model for the IR detectors, accounting for
 70 the existence of the package's internal thermal gradients. While this model was presented in [12],
 71 it is here modified to match the experimental setup used in the thermal calibration tests. Section
 72 4 describes the TIRS Thermal Mathematical Model (TMM) [13] used to define a set of theoretical-
 73 practical equations, both steady-state and transient, that allow for the estimation of the
 74 package's internal thermal gradients. Section 5 presents a detailed description of the calibration
 75 procedure, test setups carried out with the TIRS FM inside a simulation chamber, and a critical
 76 discussion of the results. Section 6 analyzes the different sources of uncertainty and potential
 77 systematic errors associated with the gradients estimation. They are used to compute a global

78 uncertainty figure for the spatial thermal gradients (section 6), which supersedes the preliminary
 79 one provided in (Table 4) of [6]. Finally, Section 7 shows the conclusions.

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82 Figure 2. General view of the TIRS calibration plan

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84 **2. TIRS thermal design and detector gradients**

85 Spatial thermal gradients in the sealed housing or package of the IR detectors arise as a result of
 86 operating in open-air applications [9], where detectors are subjected to sun irradiance [7] and
 87 wind gusts, or in applications where the entire radiometer structure is heated by an active
 88 thermal control system [8, 10, 14]. During Martian operations, the TIRS will have to operate in a
 89 thermally challenging environment. The rover Remote Sensing Mast (RSM), where the TIRS is
 90 located, is typically warmer than the Martian environment because of the rover's thermal control
 91 system. Also, the TIRS will be subjected to strong diurnal and seasonal thermal cycles, direct sun
 92 incidence and wind gusts. Thus, TIRS detectors are subject to thermal gradients on packages and
 93 these will condition the accuracy of the radiometer depending on different factors such as target
 94 view factor and filter bandpass. Thus, the narrower the filter bandpass and the smaller the target
 95 view factor, the smaller is the signal to noise ratio [15].

96 In high performance systems, and especially for channels with narrow band pass filters, the most
 97 widely used means of dealing with detector thermal gradients are mechanical choppers [8].
 98 Nevertheless, choppers require the use of mechanical actuators [16], which reduce instrument
 99 robustness and increase its energy demands and overall risk of failure. Therefore, the use of
 100 choppers is undesirable [17] for *in-situ* space radiometers. Another technical solution to thermal
 101 gradients relies on the use of active thermal control systems (cooling or warming) [8, 10, 14, 18]
 102 These systems can minimize the thermal gradients, although they introduce additional mass,
 103 power consumption and volume. Usually, they also resort to computing mechanisms to
 104 compensate for detectors' readout voltage residuals caused by the thermal gradients not
 105 completely eliminated [8, 10].

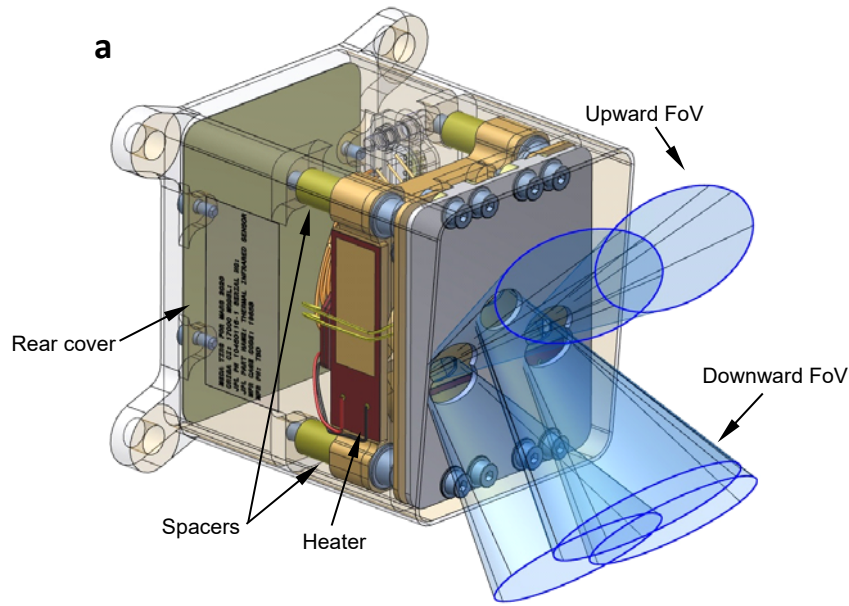
106 The thermal design of the TIRS is mainly oriented toward the minimization of the thermopile
 107 packages' thermal gradients, by using a passive thermal control system [6]. In addition to this,
 108 an off-line computing system will be used to directly estimate and compensate for the thermal
 109 gradients. The approach follow in this compensation represents an advantage compared to
 110 other proposed solutions, where the variable compensated is the detector readout voltage [7,
 111 9, 10]. Thermal gradients are independent of thermopile responsivity variations, but detector
 112 readout voltage is also affected by thermopile responsivity changes, and therefore the particular
 113 value of thermopile responsivity should be taken into account in the compensation.

114 The MEDA's Instrument Control Unit (ICU), where the TIRS electronic conditioning board is
115 located, is away from the sensor head and connected through a 3-4m long harness. In this way,
116 the thermal control to keep the electronic conditioning system within its operational
117 temperature does not thermally disturb the IR detectors, avoiding the problem of the self-
118 heating of amplification electronics and the consequent appearance of gradients described in
119 [10].

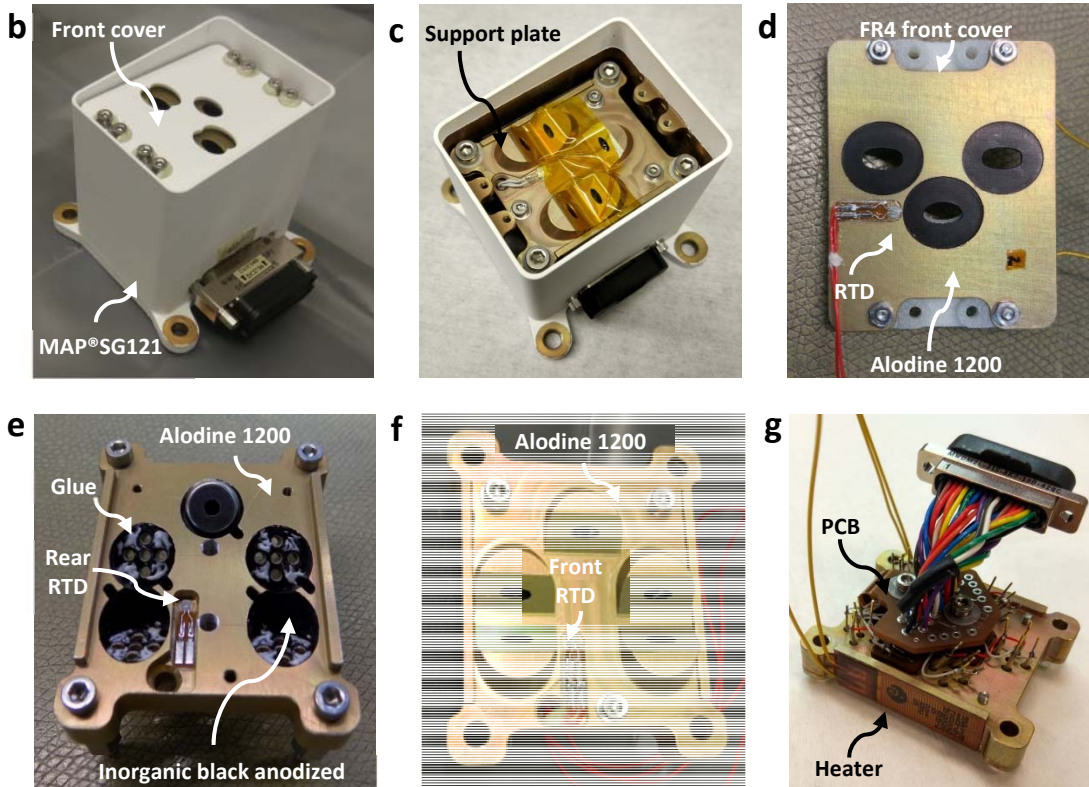
120 From a thermo-mechanical point of view, the TIRS sensor head housing consists of an external
121 aluminum structure and two fiberglass (FR4) covers, front and back (Fig.3a and b). This enclosure
122 provides a mechanical chassis to fix to the Perseverance rover RSM and protect and insulate the
123 IR detectors from the Martian environment. The external surfaces are painted using the MAP®
124 SG121 white paint, with low absorptance (≈ 0.2), to reduce the solar heat loads. All the internal
125 surfaces of the instrument have a low emissivity finish (≈ 0.08) [12], which is characteristic of the
126 chromate conversion Alodine 1200 treatment (Fig.3c, d, e, and f) thereby reducing the radiative
127 coupling between the support plate and the rest of the instrument.

128 The TIRS makes use of an almost isothermal mass, called support plate (Fig. 3c). This mass is
129 located around the thermopiles' housing to minimize the thermal gradients and allow in-flight
130 calibration [6]. The thermopiles are glued inside the support plate sub-assembly, using cryogenic
131 thermal glue MASTERBOND® EP21TCHT-1 (Fig. 3e). The support plate subassembly is formed by
132 two mechanical pieces made of aluminum that form a "sandwich" (Fig. 3f), which are
133 conductively insulated from the housing using low thermal conductivity spacers made of FR4 and
134 a flex-rigid PCB for electrical connection (Fig. 3a and g). Two Pt1000-type Resistance Temperature
135 Detectors (RTDs) glued to the rear and front mechanical structures (Fig. 4e and f), to measure
136 their corresponding temperatures T_{sp} and $T_{sp.f}$. Since it is closer to the thermopiles' package the
137 one RTD located at the rear subassembly is intended to provide the IR detectors with
138 temperature reference measurements ($T_s = T_{sp}$). Due to their different locations, there are
139 exceedingly small differences in the measured temperatures, around tens of mK. Nevertheless,
140 this difference cannot be used to estimate and compensate for the IR detector package's thermal
141 gradients because of measurement resolution. Contrary to similar designs [14, 18], the support
142 plate includes heaters only for in-flight calibration purposes, but not for thermal control of the
143 detector's temperature (thus reducing the total energy consumption). *Kapton* film heaters are
144 glued to the lateral side and are capable of providing up to 0.8W, Fig 3a and g. Heater power is
145 determined by a PI controller and a Pulse Width Modulator (PWM) driver, with the goal of
146 keeping the support plate temperature stable and above ambient temperature during in-flight
147 calibration mode.

148 The calibration plate (Fig. 3d), is also made of aluminum. The plate has holes to conform the
149 detectors FoV [6]. It acts as an in-flight calibration target, contributing to the signal seen at the
150 detector only around the periphery of the detector FoV. It induces a known signal in the detectors
151 by making use of two *Kapton* heaters of 0.8W commanded on/off and by measuring its
152 temperature (T_{cp}) with a RTD-Pt1000 contact sensor. The area of the calibration plate inner face
153 looking at the thermopiles presents an inorganic black anodized surface treatment, increasing its
154 IR emissivity above 0.9 [12]. The calibration plate is attached to the housing front cover, helping
155 to reduce the conductive coupling between the calibration plate and the rest of the sensor.



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161 Figure 3. TIRS subassemblies, coatings and Resistance Temperature Detectors (RTD) locations (a)
162 3D design drawing and FoVs, (b) The housing and front cover, (c) Internal location of the support
163 plate, (d) The calibration plate, (e) The support plate rear piece while gluing the thermopiles, (f)
164 The support plate front piece subassembly, (g) The support plate printed circuit board (PCB) and
165 electrical connections.

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3. Radiometer mathematical model

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(Eq. 1) shows the simplified mathematical model of all TIRS' channels already presented in [12].
The first term on the right-hand side (see Table 2 for the nomenclature) represents the net
radiative flux exchange with the thermopile external elements, target, and calibration and

172 support plates (Fig. 4a), through their corresponding view factors. The second term on the right-
 173 accounts for the thermopile package inner flux exchange (Fig. 4b), and it represents the effect
 174 of package thermal gradients ($T_{sf} - T_s$) calculated using the Stefan-Boltzmann law because there
 175 is no filter limiting the bandpass of the fluxes inside the thermopile. Given the operational
 176 temperatures, the IR flux leaving and coming from the support and calibration plates for a
 177 thermopile with a quartz filter (channel IR3) can be neglected, $\varphi_{cp} \sim \varphi_{sp} \sim \varphi_s \sim 0$ [6]. Therefore,
 178 (Eq. 1.1) can be rewritten into (Eq. 1.2) for this channel.

$$179 \quad V_{out} = S(T_s) \cdot A_s \cdot (F_{s-t} \cdot \varphi_t + F_{s-cp} \cdot \varphi_{cp} + F_{s-sp} \cdot \varphi_{sp} - (F_{s-t} + F_{s-cp} + F_{s-sp}) \cdot \varphi_s) \quad (1.1)$$

$$180 \quad + S(T_s) \cdot A_s \cdot (1 - F_{s-t} - F_{s-cp} - F_{s-sp}) \cdot (\sigma \cdot T_{sf}^4 - \sigma \cdot T_s^4)$$

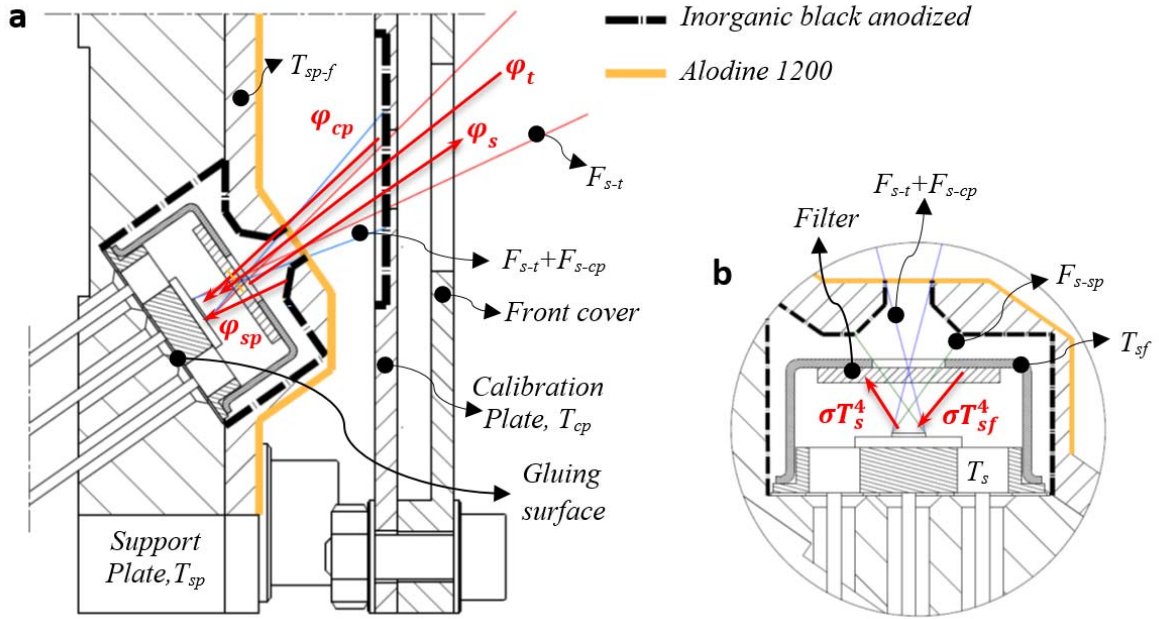
$$181 \quad V_{out} = S(T_s) \cdot A_s \cdot (F_{s-t} \cdot \varphi_t + (1 - F_{s-t} - F_{s-cp} - F_{s-sp}) \cdot (\sigma \cdot T_{sf}^4 - \sigma \cdot T_s^4)) \quad (1.2)$$

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Table 2. Nomenclature.

Parameter		Nominal values / error 1σ
V_{out}	thermopile voltage, V	
$S(T_s)$	thermopile responsivity versus temperature, V/W	
A_s	thermopile detector area, m ²	1mm ²
F_{s-sp}	thermopile detector-support plate view factor	~ 0.1815
F_{s-cp}	thermopile detector-calibration plate view factor	~ 0.0335
F_{s-t}	thermopile detector-target view factor	~ 0.0410
F_{t-sh}	target-chamber sample holder and TIRS view factor	~ 0.17±0.08
ε_t	target emissivity of the calibration set-up	0.91±0.015
φ_t	target radiosity (emitted plus reflected) inside channel spectrum, W/m ²	
φ_{cp}	calibration plate radiosity (emitted plus reflected) inside channel spectrum, W/m ²	
φ_{sp}	support plate radiosity (emitted plus reflected) inside channel spectrum, W/m ²	
φ_s	thermopile detector radiosity (emitted plus reflected) inside channel spectrum, W/m ²	
T_t	Target temperature, K	
T_s	thermopile detector temperature reference, K	
T_{sf}	thermopile detector temperature (housing front part), K	
T_{sp-f}	support plate front temperature (front piece), K	
T_{sp}	support plate temperature (rear piece), K	
T_{cp}	calibration plate temperature, K	
T_{sh}	chamber sample holder temperature, K	
σ	Stefan-Boltzmann constant, W/m ² · K ⁴	
K, h, c	Planck's law constants	
$\chi(\lambda)$	thermopile spectral response versus wavelength	
λ	wavelength, m	

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188 Figure 4. (a) Thermopiles external IR flux diagram and temperatures, (a) Thermopiles internal
189 IR flux diagram and temperatures.

190 The thermopiles' thermal gradient ($T_{sf} - T_s$) cannot be directly measured because of their small
191 value and the small size of the package, and therefore they need to be estimated. This is the
192 main objective of the thermal calibration described in this article. The thermopiles' responsivity,
193 view factors, and the detector area are known terms whose values have been previously defined
194 or subjected to radiometric and angular calibration. Their nominal and experimental values for
195 each channel are provided in (Table 2) and [12].

196 The radiosity terms for the target, detector and support plate for each channel are calculated
197 using Planck's law [12]. The expression for the calibration target has been particularized for the
198 calibration setup in section 5. The expressions for support plate and detector are shown in Eqs.
199 2.1 and 2.2. Note that a value of 1 for the support plate emissivity was considered, as it is forming
200 an almost close cavity with the thermopile package, and it is coated with a high emissivity
201 inorganic black anodized treatment[12]. Also, the emissivity of the detector is assumed to take
202 a value of 1 inside the channel bandpass because the sensing element emissivity is equal to its
203 absorptance, and its effect is already included in the channel spectral response. The calibration
204 plate radiosity calculus is more complex because of the appearance of multiple reflections, and
205 its mathematical expression can be found in [12].

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$$\varphi_s = \int_0^\infty \chi(\lambda) \frac{2\pi hc^2}{\lambda^5 \left(e^{\frac{hc}{\lambda k T_s}} - 1 \right)} d\lambda \quad (2.1)$$

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$$\varphi_{sp} = \int_0^\infty \chi(\lambda) \frac{2\pi hc^2}{\lambda^5 \left(e^{\frac{hc}{\lambda k T_{sp-f}}} - 1 \right)} d\lambda \quad (2.2)$$

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209 **4. Thermal mathematical model (TMM)**

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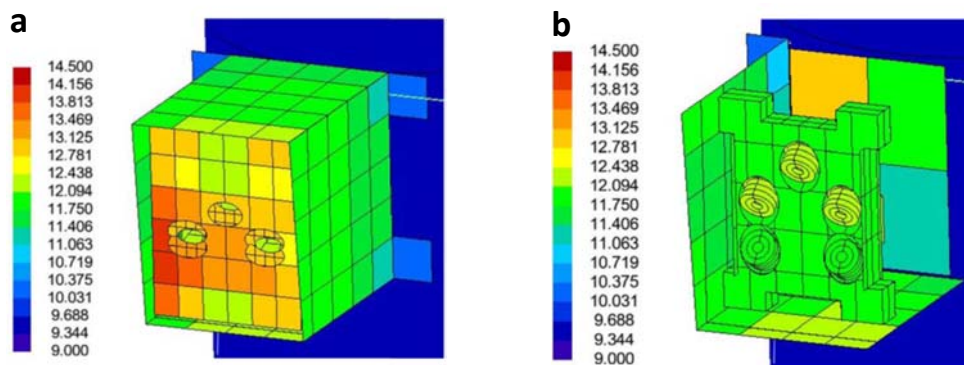
211 Thermopiles packages' thermal gradients cannot be completely eliminated by the TIRS thermo-
212 mechanical design. This is because the TIRS calibration and support plates, as well as its external
213 housing, are thermally coupled to the Martian environment, and at the same time to the
214 thermopiles' package. Therefore, the thermal loads and artifacts of the Martian environment

215 result in a thermal flux between the TIRS external housing and the thermopiles, generating
216 thermal gradients between the front and the rear elements of the thermopiles' packages [12].
217 Moreover, when the support and/or calibration plate heaters are activated during in-flight
218 calibration operational mode, additional thermal gradients will appear.

219 Since this thermal environment is very difficult and costly to recreate inside a simulation
220 chamber, here we use the TIRS's TMM [13] to: i) estimate the thermopiles' package spatial
221 thermal gradients and their correlation using estimative quantities, ii) extend the validity of the
222 calibration test results to the different operational scenarios (temperature range, wind and solar
223 loads), and iii) help define of the accuracy of the thermal gradient estimators.

224 The TIRS's TMM is based on geometrical mathematical models and use the software tool
225 ESATAN-TMS r7 released in 2017. The TMM consists of 823 thermal nodes, 820 of which are
226 shell nodes (3D geometrical finite elements) and 3 are non-geometrical nodes (external and
227 internal atmosphere finite elements). The nodes represent the conductive and radiative
228 boundaries of the instrument. Radiative calculations have been performed with Monte Carlo
229 Ray Tracing. Internal conductive couplings have been calculated considering the thermal contact
230 resistance. Convection has also been considered for small internal cavities, where the effects of
231 the Martian atmosphere are not negligible. The TMM has been completed with the addition of
232 the solar loads, the convective exchange factor calculation, and the boundary conditions. In
233 addition, a reduced model of the NASA Mars 2020 rover has been used to represent the radiative
234 environment of the instrument with a higher fidelity, including the effect of the shade produced
235 by the rover, which can be an important factor for the gradients, due to the sudden change in
236 solar loads. The rover yaw angle has been selected to represent a worst orientation for each
237 case. The orientation that produces maximum solar irradiation during all the day has been
238 applied for the hottest day (except when it is shadowed by the rover). On the other hand, the
239 selected orientation for the coldest day leads to the instrument to be protected from solar
240 irradiation during most of the time. The analyzed scenarios are the diurnal cycle for the
241 simulated coldest day or WCC (worst cold case), and hottest day or WHC (worst hot case). These
242 two extreme cases cover the whole range of temperature conditions the instrument will
243 encounter, ensuring the existence of changes in the solar load seen by the TIRS.

244 The need to verify the thermal gradients of the thermopiles package leads to a detailed
245 representation with higher discretization in the number of thermal nodes of the thermopile
246 surrounding elements, despite the high thermal conductivity of those components made of
247 aluminum (Fig. 5). The package of the thermopiles has been modeled by two thermal nodes,
248 one at the front and the other at the rear, with the aim of evaluating the thermal gradient
249 between the two. The thermal properties of the package material were defined according to the
250 base material, polished stainless steel, with an IR emissivity of ~ 0.1 [12]. This represents a
251 relevant simplification since the filter and its thermal properties have not been modeled.



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253 Figure 5. TIRS TMM temperature map results in Celsius for the WHC (Worst Hot Case) scenario
254 and heaters OFF. (a) The front cover and housing, and (b) the support plate and thermopiles.

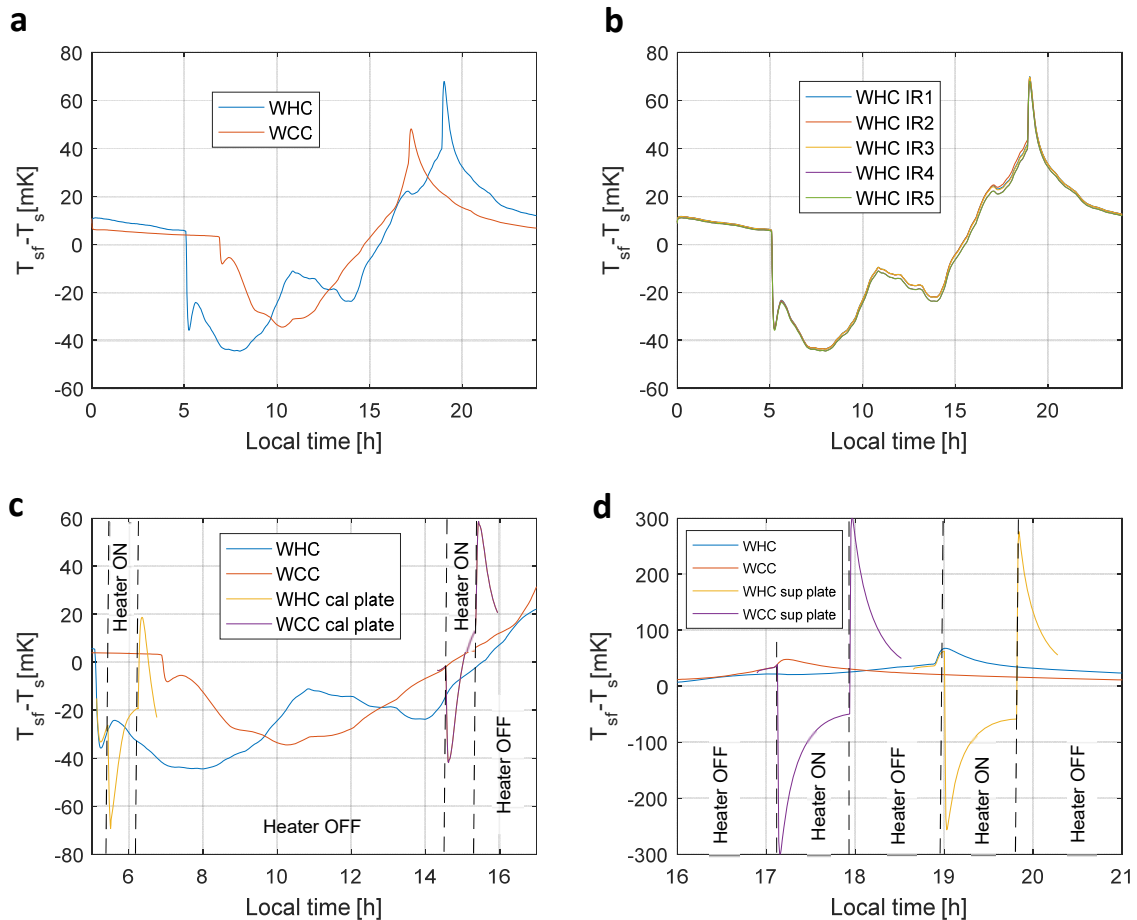
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256 Finally, to reduce uncertainty in the TMM results, some of the model parameters have been
257 adjusted and correlated following the procedure described in [19], with the quasi-steady and
258 transient results of dedicated tests. The parameters of the TMM that were modified during the
259 correlation are five: The conductive couplings between TIRS housing and the support and
260 calibration plates through the bolted joints and cabling, the conductive couplings between
261 support plate and the thermopiles through gluing and cabling, and finally the thermopile
262 housing thermal capacitance. The tests were performed in a simulation chamber recreating
263 Martian atmospheric pressure and CO₂ composition. We employed a TIRS engineering model
264 and three contact temperature sensors to monitor boundary conditions for chamber,
265 mechanical interface and TIRS housing temperatures. The thermopile package thermal gradients
266 were estimated using a quartz filter thermopile, IR3 channel, based on the simplified thermopile
267 mathematical model for this channel (Eq. 1.2). The chamber was kept in dark conditions, $\varphi_t = 0$,
268 so that thermopile voltage was only due to package gradients. The tests consisted of the
269 activation of TIRS calibration and support plate heaters in different sequences, while
270 temperatures were logged with a sampling rate of 90 sec. In the first test, calibration plate
271 heater was power on during 1h with 0.8W, after that support plate heater was activated during
272 4 additional hours at maximum power, 0.8W. In the second test, the power of support plate
273 heater was determined by the PI controller for four different support plate reference
274 temperatures references, that were changed every hour. The calibration plate heater remained
275 off during this test. After TMM parameters fit, the correlation with experimental temperatures
276 of the TIRS mechanical elements was better than 0.6K, while the correlation with the gradients
277 in the thermopile package was better that 5mK.

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279 4.1. Thermal model results

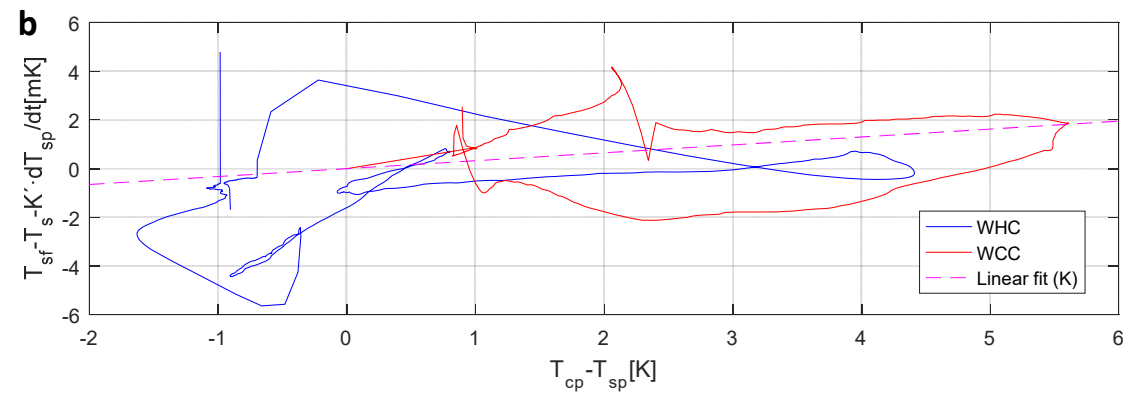
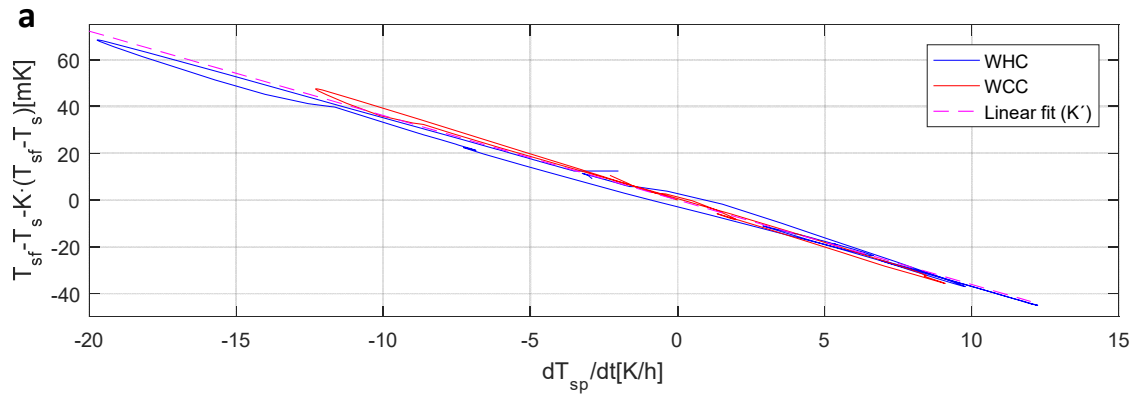
280 The TMM results for the temporal evolution of the thermopiles' package thermal gradients are
281 presented here. (Fig. 6a) shows the evolution of the thermal gradients for channel IR3 under
282 both scenarios, while (Fig. 6b) shows the same for the other channels under the WHC. Note that
283 (Fig. 6b) shows no significant difference in the gradients of the different channels, which
284 highlights the almost null effect of the thermopiles particular location and orientation.
285 Additional cases where calibration and support plate heaters were dissipating 0.8 W during 30
286 and 50 min have been simulated. (Figs. 6c and 6d) show the results, with clear effects in the
287 thermopile package gradients as a result of the driven power. In the case of the calibration plate
288 heater activation (Fig. 6c), the generated gradient after the initial transient was positive during
289 the heating (front part of the thermopile warmer than the rear). Conversely, the support plate
290 heaters generate a negative gradient during the heating (Fig. 6d), which is compatible with the
291 way the heat reaches the thermopiles from the rear part of the support plate.



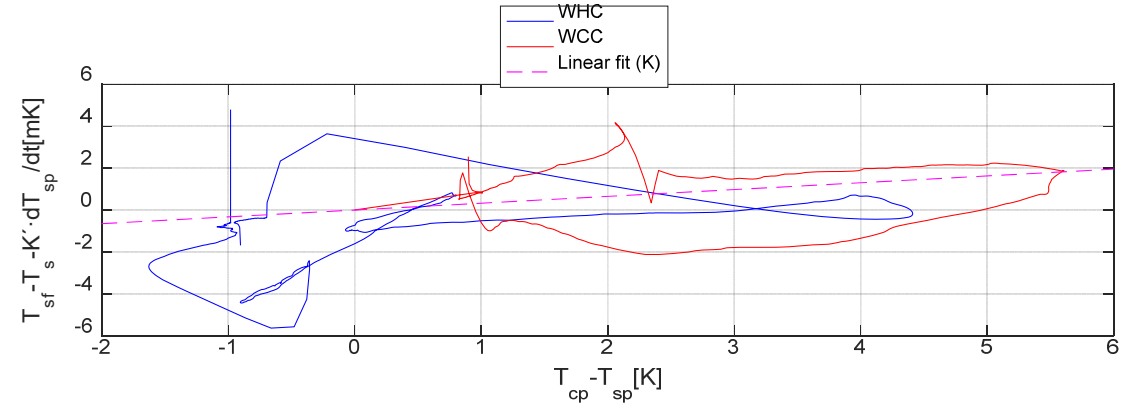
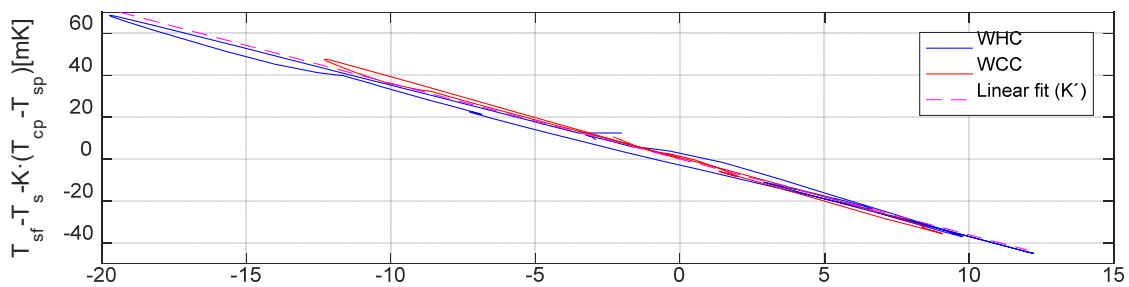
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 293 Figure 6. TMM results for thermal gradients between package front and rear. (a) Diurnal
 294 evolution of the IR3 channel for WHC and WCC scenarios, (b) Diurnal evolution of all channels
 295 for the WHC scenario, (c) The IR3 channel during calibration plate heating for WHC and WCC
 296 scenarios, (d) The IR3 channel during support plate heating for WHC and WCC scenarios.

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 298 A detailed analysis of the gradients during the diurnal WHC and WCC shows a high degree of
 299 correlation between the time derivative of the support plate temperature and the temperature
 300 gradient in the thermopile package, as shown in (Fig. 7a). Although not shown in this figure, this
 301 correlation works fine for the rest of the channels too. For each temperature time derivative,
 302 the temperature gradient in the thermopile package takes two slightly different values, due to
 303 the passage through said time derivative during the diurnal cycle increasing or decreasing the
 304 value of the derivative. Also, there is a difference between the WHC and WHC, that may be
 305 caused by the change in the model boundary conditions for both cases. These two effects shows
 306 that the gradients in the packages are also affected by other quantities, such as the existence of
 307 spatial gradients in the sensor structure (ex. between support and calibration plate), which are
 308 controlled by the external temperature evolution and solar loads. The effect of these spatial
 309 gradients between the calibration and the support plates can be seen in (Fig. 7b). Although the
 310 correlation is not as clear as in the case of the temporal derivative, a certain dependence may
 311 be observed.

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319 Figure 7. IR3 gradients analysis, (a) Correlation between support plate temperature time
 320 derivative and thermopile's package thermal gradient, (b) Correlation between calibration
 321 and support plates temperature difference and thermopile's package thermal gradient.

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323 *4.2 Thermopiles' thermal gradients estimators*

324 Based on the results of the TMM, a set of mathematical equations has been defined as estimators
 325 to compensate for the thermopiles' thermal gradients effects. A similar compensation approach
 326 was already described in [8], where an empirical time dependence relation was established
 327 between the variation of the detector package temperature and the error induced in the detector
 328 output voltage. Also, in [10, 14, 18] a steady state relation between the power driven by the
 329 radiometer active thermal control system and the output voltage of the detectors was used with
 330 similar objectives. Contrary to these two approaches in which estimators compensate for the
 331 error output voltage of the detector, TIRS thermal estimators compensate for the package
 332 thermal gradients of the detector. This represents an advantage because this is the real
 333 mechanisms in which the error signal is generated, and therefore it is not affected by thermopile
 334 responsivity variation versus temperature. This allows for a simplification of the calibration setup
 335 and tested temperatures.

336 The design of the TIRS thermal gradient estimators accounts for operational modes. For nominal
 337 operation, the thermal gradients created by the diurnal temperature evolution and artifacts
 338 such as shadows and wind gusts are estimated using a linear and time dependent function
 339 shown in (Eq. 3.1). In this function, the thermopile spatial gradient is calculated from the
 340 temperature difference between the calibration and support plates, and the support plate
 341 temperature time derivative, through the constants K and K' respectively. The reason for
 342 selecting the spatial gradient between the support and calibration plates, instead of the gradient
 343 inside the support plate, which should be more related to the thermopiles' gradient, is that the
 344 former can be more easily measured because its value is higher and it is less affected by the
 345 resolution of the measuring electronics. (Fig. 7a and 7b) show the results of fitting of the
 346 proposed estimator for the data of the TMM (linear fit of K and K'). To graphically assess the fit,
 347 the y-axis of the figures has been modified to subtract the contribution of the other fitting term.
 348 The fit provides values of $K_{TMM} = 0.325 \text{ mK/K}$, $K'_{TMM} = -3.612 \text{ mK/K} \cdot h$, with standard
 349 errors of $\sigma_{K_{TMM}} = 0.015 \text{ mK/K}$ and $\sigma_{K'_{TMM}} = 0.005 \text{ mK/K} \cdot h$, representing error of 4.61%
 350 and 0.14% respectively, and a standard deviation of the misfit of 1.38 mK.

$$351 \quad T_{sf} - T_s = K \cdot (T_{cp} - T_{sp}) + K' \cdot \partial T_{sp} / \partial t \quad (3.1)$$

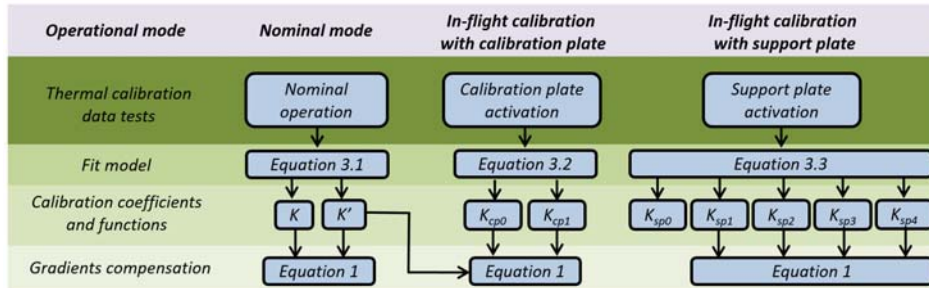
$$352 \quad T_{sf} - T_s = K_{cp0} + K_{cp1} \cdot (T_{cp} - T_{sp}) + K' \cdot \partial T_{sp} / \partial t \quad (3.2)$$

$$353 \quad T_{sf} - T_s = K_{sp0} + K_{sp1} \cdot Power_{sp} + K_{sp2} \cdot (T_{cp} - T_{sp}) + K_{sp3} \cdot Power_{sp}^2 + K_{sp4} \cdot Power_{sp} \cdot (T_{cp} - T_{sp}) \quad (3.3)$$

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 355
 356 Outside the nominal operations, in-flight calibration operation modes are also affected by
 357 thermal gradients in the thermopiles. For the in-flight calibration mode using the calibration
 358 plate, the calibration plate heating power is constant but the generated thermopile package's
 359 gradient during its activation is modified by external factors such as: the external thermal
 360 scenario, solar loads, and winds. Therefore, to take into account this fact the thermopile package
 361 gradients estimation function linearly depends on the spatial gradient between the calibration
 362 and support plates (Eq. 3.2). This spatial gradient has been selected because it is strongly
 363 affected by heater activation and depends on previously described external factors. The in-flight
 364 calibrations will be executed during the period of the Martian day when ambient and ground
 365 temperatures are more stable (ex. before sunrise or at noon). This will help to limit the support
 366 plate temperature time derivative, but without making its value null. Therefore, the time
 367 derivative of the support plate is also part of the estimator for this calibration mode (Eq. 3.2).
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370 For the calibration mode using the support plate, a steady state function is defined (Eq. 3.3),
 371 because the support plate temperature is kept stable by the PI controller and the term
 372 depending on K' is not needed. The equation shows a two variables nonlinear function, assuming
 373 a polynomial of order 2 for power driven to the support plate heater since, in this case, heating
 374 power is modulated by the PI controller, and order 1 for the spatial gradient between the

375 support plate and the calibration plate. The spatial gradient between the calibration and support
 376 plates has been selected because of reasons like those exposed above. The independent term
 377 of the polynomial is assumed to take a null value, $K_{sp0} = 0$, since the gradients in the package
 378 must be null in the absence of power dissipated in the support plate. (Fig. 8) summarizes the
 379 thermal gradient calibration and compensation approach.
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381
 382 Figure 8. Thermal gradients estimation and compensation approach.

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384 5. Thermal calibration tests

385 The TIRS thermal calibration plan includes a set of tests to identify the value of the constants
 386 and functions of the thermopiles' thermal gradients estimators appearing in Eqs. 3.1, 3.2 and
 387 3.3.

388 5.1 Experimental setup

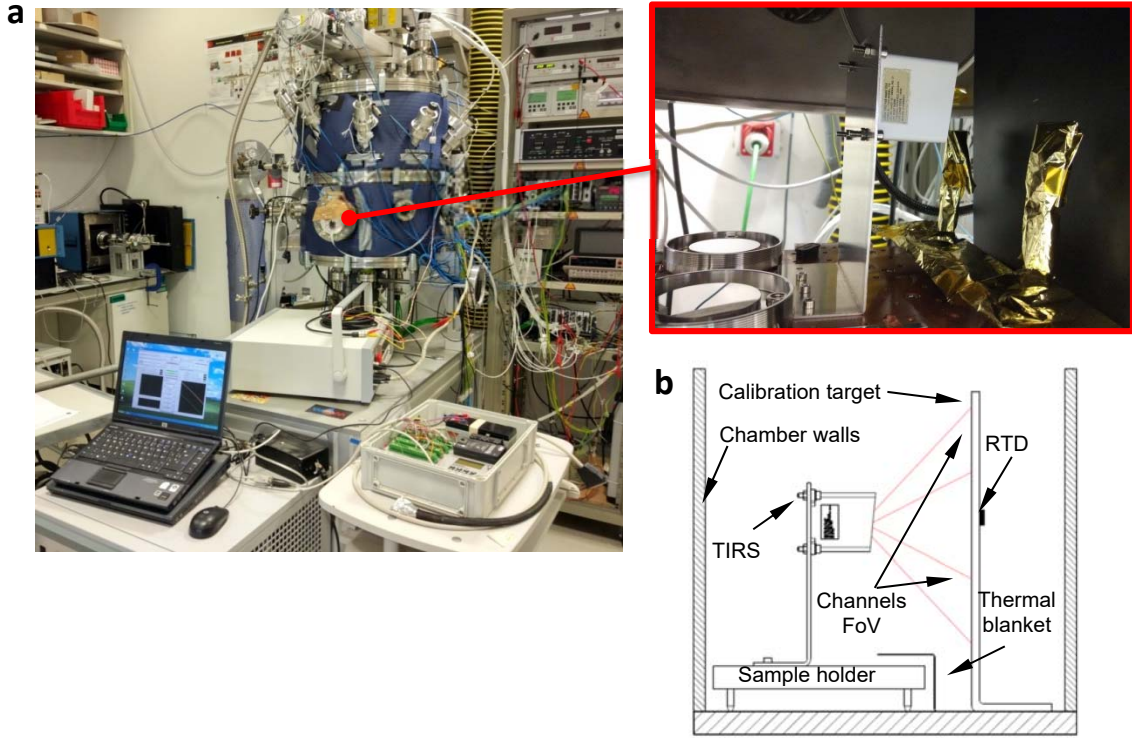
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390 The MARTE simulation chamber [20], designed by the Centro de Astrobiología in Madrid (Spain),
 391 has been used as the main testing facility (Fig. 9a). This facility complies with the requirements
 392 and limitations imposed by the planetary protection plan of the mission. The Martian
 393 atmospheric composition and pressure is recreated inside the chamber, evacuate down to 10^{-3}
 394 mbar and then filling the chamber using as base gas a bottle of 100% of CO₂ till reaching
 395 8mbars. The pressure is measured with a capacitive temperature compensated sensor model
 396 CMR 232 from Pfeifer GmbH.

397

398 The chamber allows the simulation of the gradient between the rover mast and the Martian
 399 atmosphere and its diurnal evolution. To this end, the TIRS was thermally coupled to chamber
 400 sampler holder. The sample holder has the capacity to generate temperature profiles within
 401 $\pm 20K$ around room temperature, while the chamber walls remain at this room temperature. This
 402 is the main limitation of the setup, since it is not possible to cover the entire TIRS operational
 403 temperature range [13]. Another limitation is the inability to recreate the effects of wind gusts
 404 or solar loads, due to the difficulties to generate wind flow at Martian pressure and the influence
 405 of both effects in the temperature stability of the calibration target.

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Figure 9. MARTE chamber test setup, (a) External and internal photos, (b) Simplified calibration diagram

416 Inside the chamber there is a calibration target formed by a copper plate painted with a high
417 emissivity coating (Aeroglaze® Flat Black Absorptive Polyurethane with an average emissivity of
418 ~ 0.91 in the range $5\text{-}35\ \mu\text{m}$) [21]. The target was thermally coupled to the chamber walls (Fig.
419 9b). Its temperature was not controlled but measured using a RTD glued to its surface. The
420 chamber and the calibration target form a closed cavity at a similar temperature. Nevertheless,
421 the chamber sample holder is awfully close to the calibration target and the changes in its
422 temperature during the calibration generate reflections on its surface that modify the
423 calibration target radiosity. To compensate for this effect, a simplified geometric model of the
424 calibration setup has been implemented. The model allows for computing the radiosity of the
425 calibration target φ_t by integrating Planck's law inside the channels' bandpass from the
426 measured temperatures of the target and the sample holder, the emissivity of the target radiant
427 surface and theoretical values of the vision factors between the focused surfaces of the
428 calibration target and the walls of the chamber and the surface of the sample holder (Eq. 4). In
429 the case of channel IR3, considering the bandpass of the quartz filter, the test is performed in
430 dark conditions, $\varphi_t \sim 0$.

$$\begin{aligned}
 431 \quad \varphi_t = & (\varepsilon_t + (1 - \varepsilon_t) \cdot (1 - F_{t-sh})) \cdot \int_0^\infty \chi(\lambda) \frac{2\pi hc^2}{\lambda^5 \left(e^{hc/\lambda K T_t} - 1 \right)} d\lambda \\
 432 & \\
 433 \quad & + (1 - \varepsilon_t) \cdot F_{t-sh} \cdot \int_0^\infty \chi(\lambda) \frac{2\pi hc^2}{\lambda^5 \left(e^{hc/\lambda K T_{sh}} - 1 \right)} d\lambda
 \end{aligned} \tag{4}$$

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Nominal values for F_{t-sh} and ε_t , as well as their associated uncertainties, are taken from (Table 2). Target and chamber sample holder temperatures are logged during the tests. The associated temperature uncertainties are ± 0.2 K (standard error) for the target, including RTD and target contact temperature inhomogeneity, and ± 1 K for the chamber sample holder. The latter is larger because of the thermal gradients affecting the sample holder. This leads to an uncertainty in the target radiosity equivalent to ± 0.3 K.

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Electrical measurements such as the calibration target, the support and calibration plate temperatures, the thermopiles' voltages and the support plate heater current, were performed using a KEITHLEY 2700 multimeter and a KEITHLEY 7700 commutation matrix. The multimeter was configured to log the value for all the temperatures and voltages every 30 seconds. Each particular sample is calculated by averaging 100 readings and by using an analog filter to reject 50 Hz noise. The heater driver is made up of a constant 12 V power supply and a ~~PWM power wide modulator~~ to control the driving voltage for the case of the support plate driver. The modulator duty cycle was adjusted to provide 0, 25, 50 and 100% of the maximum power. The power dissipated at the heaters is calculated by measuring the current passing through them, assuming the nominal value of heaters' resistance. The power estimation error is below 2.8%, negligible for calibration purposes and mainly determined by current measurement error and heater resistance tolerances.

455 **5.2 Calibration tests results**

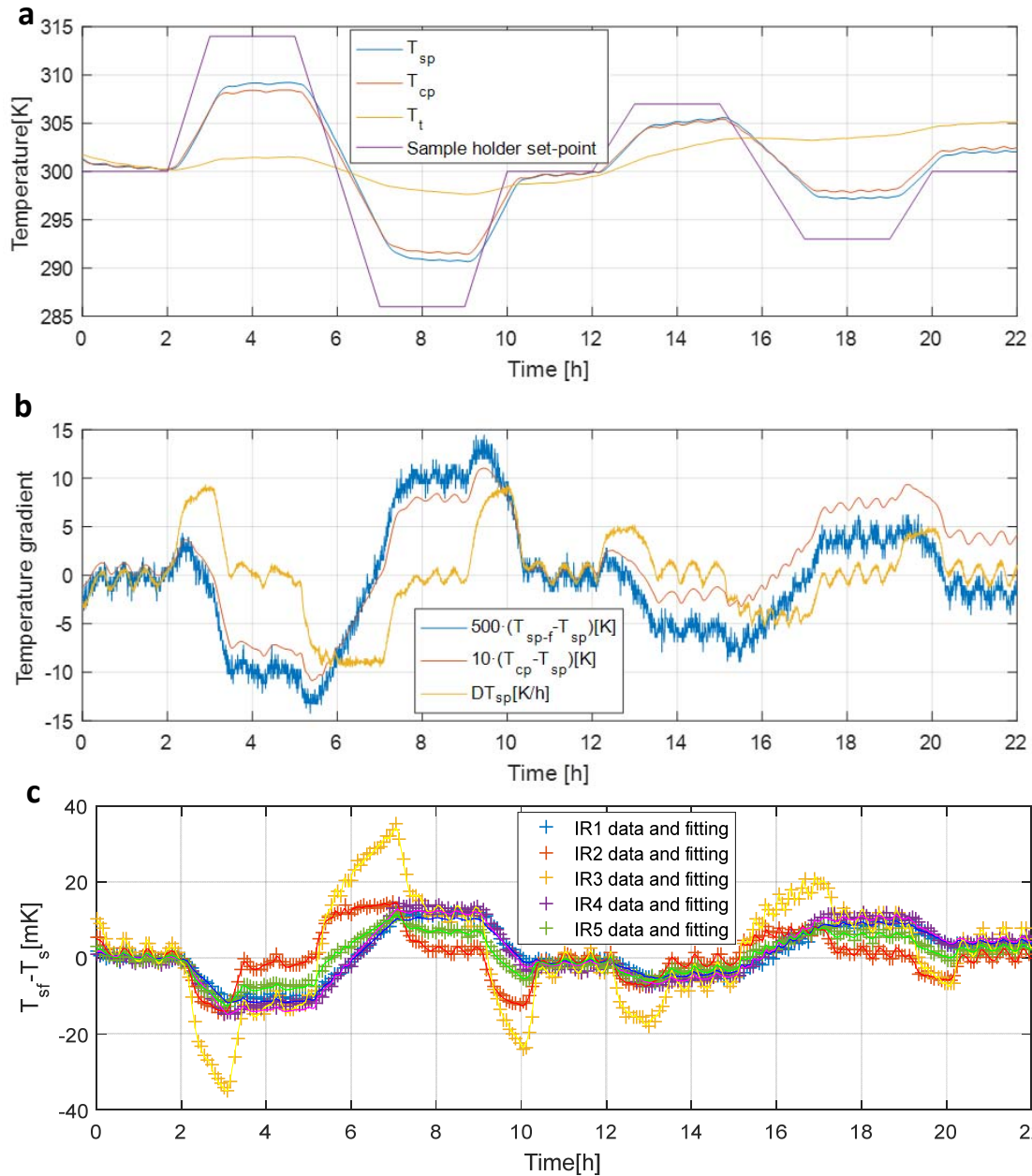
456 The experimental thermal calibration consisted of two different sets of tests, recreating the
457 nominal and the in-flight calibration operational modes (Table 3).

Test purpose	Target temperature [K]	Sample holder [K]	Support plate heater power [W]	Calibration plate heater power [W]	Test conditions
1. Nominal mode (<i>K</i> and <i>K'</i> identification)	~300	286, 293, 300, 307, 314 Profile: ±14K/h, ±7K/h	0	0	Mars atmosphere, 8 mbar
2. In-flight calibration modes (Heaters function identification)	~300	286, 293, 300, 307, 314	~0, 0.2, 0.4, 0.8	~0, 0.8	Mars atmosphere, 8 mbar

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Table 3. TIRS thermal calibration tests and conditions

The test for the nominal mode was aimed to simulate the thermal environment on Mars, while measuring the evolution of the thermopiles' thermal gradients. It consisted of creating a profile with different temperatures in the chamber sample holder (Fig. 10a). The change in the sample holder temperature, together with a smooth evolution of the temperature of the chamber's inner atmosphere, simulate the existence of positive and negative temperature differences (±14K and ±7K) between the rover RSM and the Martian atmosphere and generates thermal gradients between support and calibration plates, $T_{cp} - T_{sp}$. The sample holder temporal gradients (14K/h and 7K/h) simulate the diurnal evolution and the change rate of support plate temperature, $\partial T_{sp} / \partial t$ (Fig. 10b). These figures have been taken from the results of the TMM, that includes expected Martian conditions and the effect of solar loads and wind gusts [13].



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472 Figure 10. Nominal mode calibration test, (a) Chamber set point and TIRS temperatures profile,
 473 (b) Thermal gradients in TIRS subassemblies, (c) Thermopiles' package thermal gradient derived
 474 from (Eq. 5 and 5.2), decimated samples (+) and model fit (solid lines) derived from (Eq. 3.1).

475 Practical test results (Fig. 10b), show that the expected values for support plate time derivative
 476 has been partially achieved, with ± 10 K/h versus the $+10$ K/h and -20 K/h for the TMM, while the
 477 test was not capable of covering Martian range for $T_{cp} - T_{sp}$. Based on the results of the TMM
 478 $T_{cp} - T_{sp}$ should take values from -1.5 K to $+5.6$ K, while in the practical test is limited to ± 1 K. The
 479 main reason is the impossibility to recreate solar loads.

480 Measured data are introduced in (Eq. 1.1 and 1.2) together with the previously calibrated
 481 constant and terms of [12]. Then (Eq. 1.2) for channel IR3, and (Eq. 1.1) for the other channels
 482 are solved for T_{sf} , obtaining (Eq. 5.2 and 5.1) respectively. Finally, the evolution of the
 483 thermopile package gradient ($T_{sf} - T_s$) is computed (Fig. 10c).

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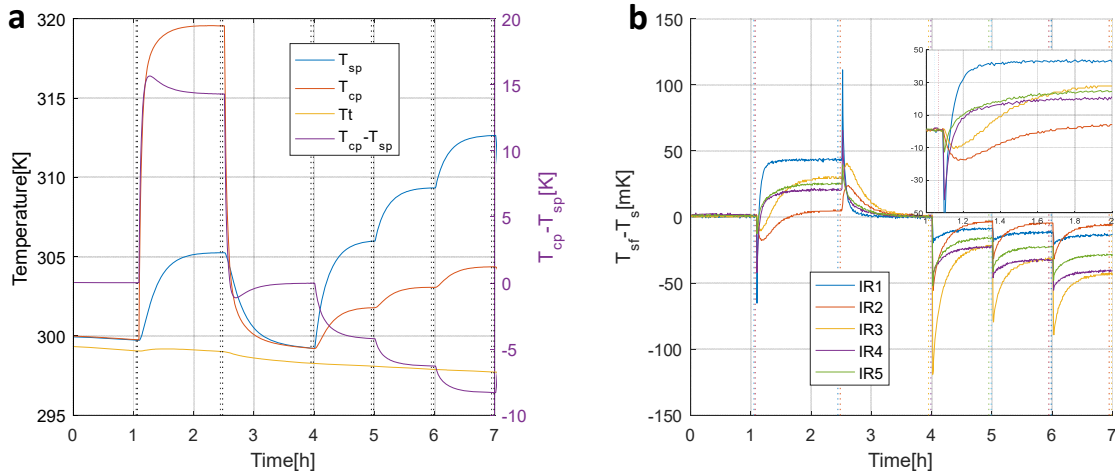
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$$T_{sf} = \left(\frac{\left(\frac{V_{out}}{S(T_s) \cdot A_s} - F_{s-t} \cdot \varphi_t - F_{s-cp} \cdot \varphi_{cp} - F_{s-sp} \cdot \varphi_{sp} + (F_{s-t} + F_{s-cp} + F_{s-sp}) \cdot \varphi_s \right)}{\left((1 - F_{s-t} - F_{s-cp} - F_{s-sp}) \cdot \sigma \right)} + T_s^4 \right)^{1/4} \quad (5.1)$$

485
$$T_{sf} = \left(\frac{\left(\frac{V_{out}}{S(T_s) \cdot A_s} - F_{s-t} \cdot \varphi_t \right)}{\left((1 - F_{s-t} - F_{s-cp} - F_{s-sp}) \cdot \sigma \right)} + T_s^4 \right)^{1/4} \quad (5.2)$$

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 487 The experimental values obtained for the thermopiles' package gradients are different
 488 depending on the channel (Fig. 10c). For some channels, particularly IR2 and IR3, the gradients
 489 introduce important uncertainties, potentially compromising the accuracy requirements [6], and
 490 thus justifying the need for calibration presented in this article (Table 4). Equivalent target
 491 temperature or irradiance errors have been calculated assuming a nominal thermal
 492 environment [6], where ground temperature is equal to $T_s - 20K$, sky temperature is $T_s - 70K$ and
 493 atmosphere temperature is $T_s - 10K$.
 494

Channel	Maximum gradient [mK]	Equivalent error $T_s = 298K$	Equivalent error $T_s = 173K$
IR1	±12	±2.55W/m ^{2a}	±2.37W/m ^{2a}
IR2	±15	±13.77K	±13.44K
IR3	±35	±4.49W/m ²	±4.49W/m ²
IR4	±14	±2.72W/m ^{2a}	±2.72W/m ^{2a}
IR5	±11	±0.71K	±0.60K

495 ^a This error is calculated assuming a blackbody model for the target and Stefan-Boltzmann emission
 496 Table 4. Nominal mode experimental values for package's gradients and equivalent target
 497 temperature or irradiance uncertainties without gradients compensation for two different
 498 thermopile package temperatures.
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500
 501 Figure 11. Thermopiles' package gradients results for in-flight calibration modes and chamber
 502 sample holder temperature of ~300K. (a) Temperature profiles vs. time during test
 503 development. (b) Thermopiles' package thermal gradients derived from (Eq. 5.1 and 5.2) vs.
 504 time, and (insert) zoom during calibration plate heating.
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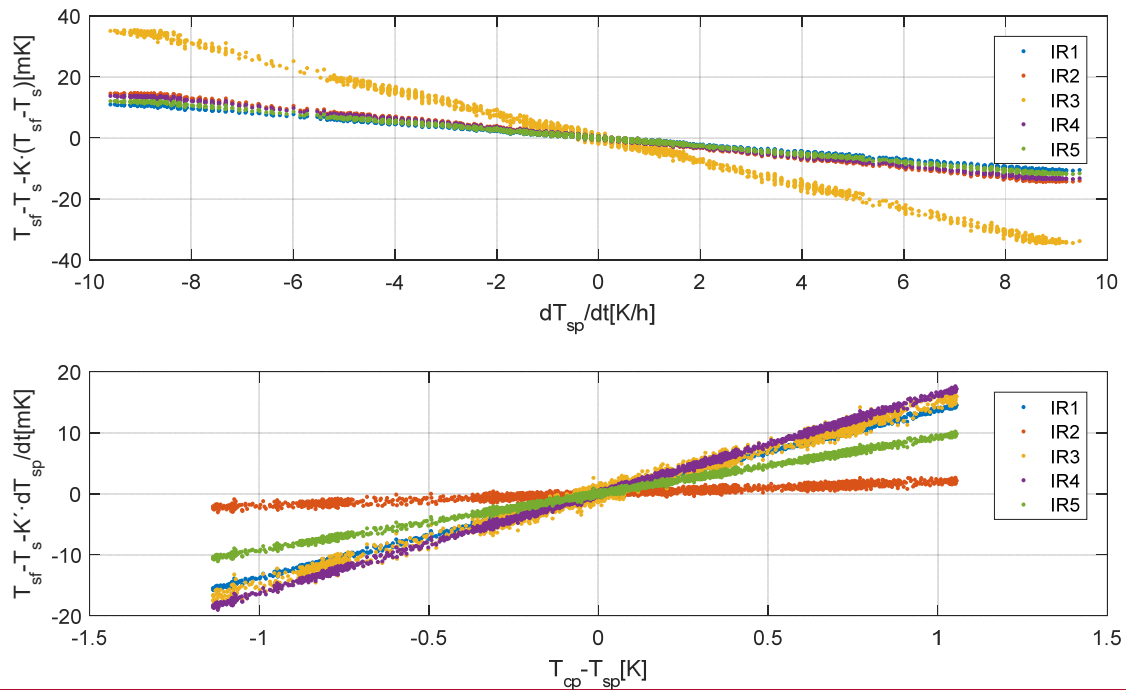
506 The aim of the second set of tests was to simulate the execution of the in-flight calibration
 507 algorithm either by driving power to the calibration plate or support plate, while measuring the
 508 evolution of the thermopiles' thermal gradients. The chamber sample holder temperature was
 509 set to different values (Table 3). For each temperature, a test sequence first drives power to the
 510 calibration plate during 1.5h, and then during 3h to the support plate with three different power
 511 levels: 25, 50 and 100% (Fig. 11a). A 1.5h of stabilization time is allowed between heater

512 activations to reach steady state conditions. Then, (Eq. 5.1 and 5.2) are solved, and then the
513 thermopile gradient ($T_{sf} - T_s$) is computed (Fig. 11b).

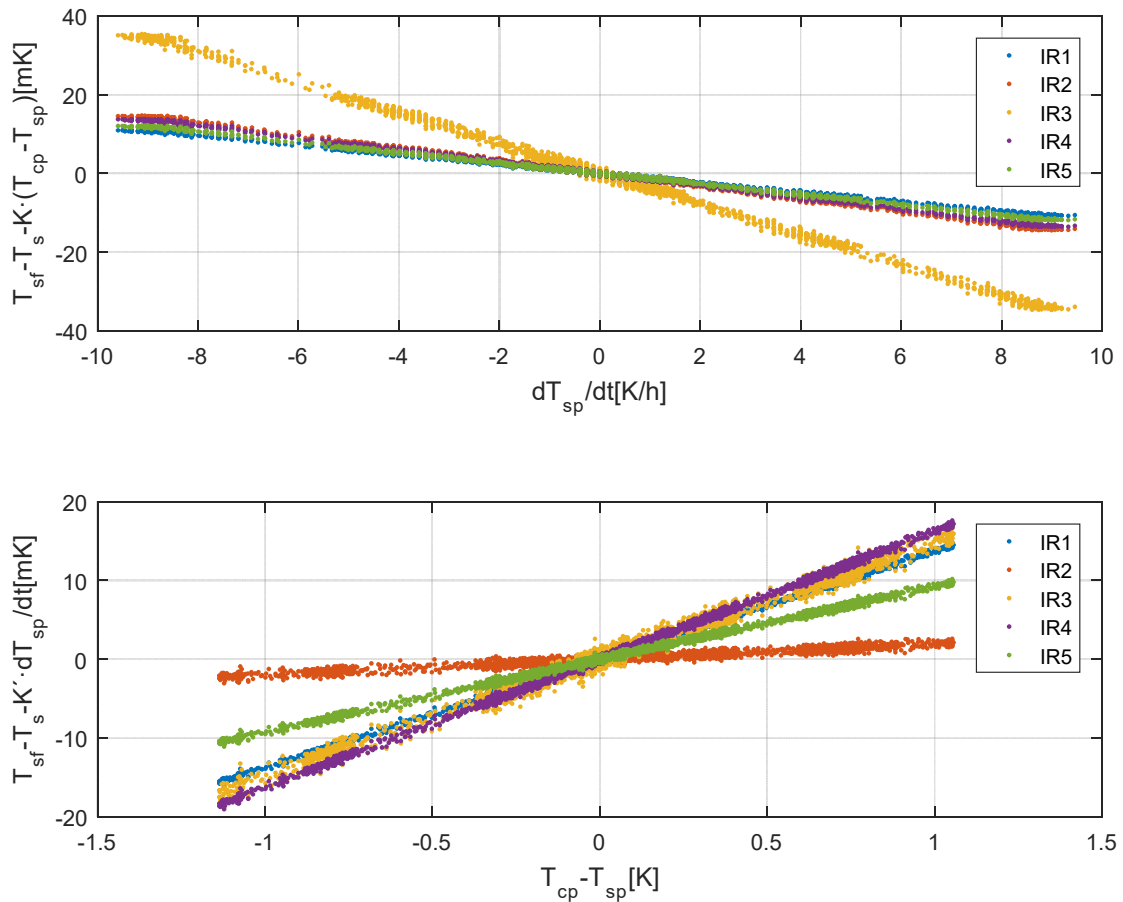
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515 5.3 Estimators fit

516 Nominal mode test gradients were fit according to (Eq. 3.1) using a least square algorithm [22].
517 (Fig. 12) shows the results of the fit in the same way as for the TMM data. In this case, a clearer
518 correlation with calibration and support plate temperature gradient is seen (Fig. 12b). The
519 estimated coefficients, K and K' , their standard error σ_K and $\sigma_{K'}$, and fit residuals are provided
520 in (Table 5). (Fig. 10c) also shows the results of the fit (solid lines). The results of the different
521 channels differ among them, but the results are consistent with the expected value, and largely
522 coincided with the results of other models such as the Flight Spare (FS) and Bread Boards (BB
523 and BB1).



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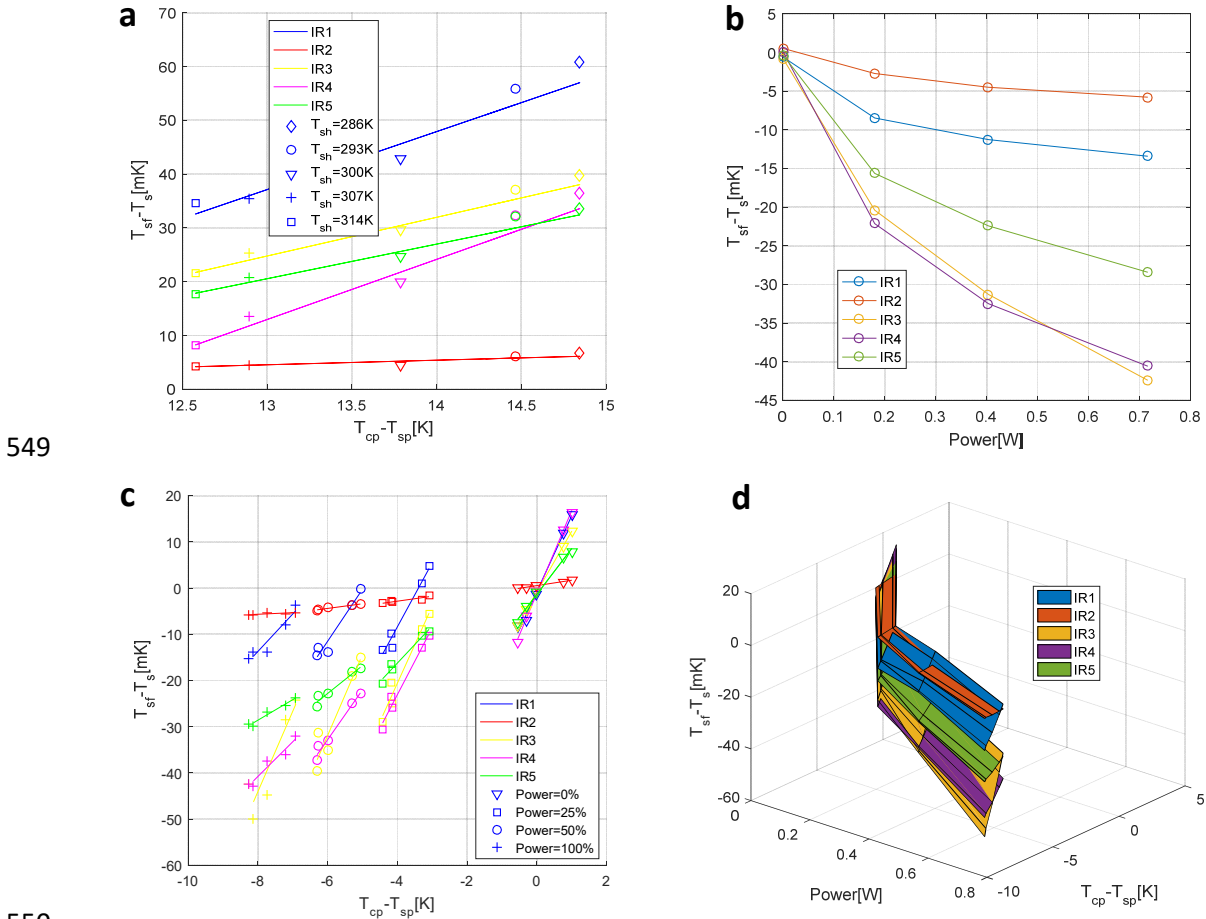
Figure 12. Nominal mode practical gradients analysis, (a) Correlation between support plate temperature derivative and thermopile's package thermal gradient (K' constant), (b) Correlation between calibration and support plates temperature difference and thermopile's package thermal gradient (K constant).

Channel	K [mK/K]	σ_K [mK/K]	K' [mK/K/h]	$\sigma_{K'}$ [mK/K/h]	Fit residual- 1σ [mK]
IR1	13.70	0.0318	-1.205	0.0042	0.919
IR2	1.93	0.0095	-1.602	0.0012	0.438
IR3	14.91	0.0268	-3.861	0.0035	0.771
IR4	16.21	0.0322	-1.513	0.0042	0.929
IR5	9.21	0.0180	-1.326	0.0024	0.584

533 Table 5. Nominal mode gradient estimation coefficients (K and K'), associated estimation
534 uncertainties (σ_K and $\sigma_{K'}$) and fit residuals

535 The test results for the calibration modes are summarized in (Fig. 13). For these modes only
536 steady state gradients during calibration test development were considered, which are marked
537 with vertical dotted lines in (Fig. 11a and b). (Fig. 13a) shows a compilation of the experimental
538 values for the thermopiles' package gradients versus the spatial gradient between the support
539 and calibration plates for all sample holder temperatures during the heating of the calibration
540 plate. In addition, linear fits of the data for each channel [22], according to the gradients'
541 estimation function of (Eq. 3.2), are plotted. Since this test uses data once the steady state
542 regime has been reached, the term of (Eq. 3.2) depending on the temperature time derivative
543 of the support plate takes a null value, and the associated coefficient cannot be identified. Thus,

544 the time derivative coefficient from (Eq. 3.1) is applicable for (Eq. 3.2) and the gradients
 545 compensation in this calibration mode. (Table 6) provides the coefficients of estimation
 546 polynomials and the fitting error. Standard errors for the coefficients have not been provided
 547 because of the small number of data (5 points) that leads to high and not real uncertainties. Fit
 548 residuals are compliant with target radiosity uncertainty.



550
 551 Figure 13. Thermopiles' package gradients results and fit for in flight calibration modes (a)
 552 Experimental data and linear estimation functions of the thermopiles' package gradients during
 553 the calibration using the calibration plate, (b) Experimental data of thermopiles' package
 554 thermal gradients versus heating power during the calibration using the support plate for a
 555 chamber sample holder temperature of ~300K, (c) Experimental data and linear estimation
 556 functions of thermopile's thermal gradients for the different heating powers during the
 557 calibration test using the support plate, (d) Polynomial surface estimation functions of
 558 thermopile's thermal gradients for the calibration mode using the support plate heater.

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Channel	K_{cp0} [mK]	K_{cp1} [mK/K]	Fit residuals-1 σ [mK]
IR1	-102.9	10.77	4.8
IR2	-6.5	0.85	0.66
IR3	-68.9	7.20	2.6
IR4	-132.3	11.17	4.3
IR5	-63.1	6.43	2.4

561 Table 6. Gradients estimator and fit residuals for the calibration mode using the calibration plate
 562 (Fig. 13b) shows the quadratic dependence of the thermopiles' package gradients versus the
 563 heating power during support plate heating. (Fig. 13c) does the same for the linear dependence

564 of the package gradients versus the thermal gradient between the support and calibration plates
 565 for the different heating powers. Finally, (Fig. 13d) shows a 3D plot with the results of the global
 566 non-linear gradient estimation functions according to (Eq. 3.3). This function has been obtained
 567 by means of a polynomial adjustment using the Matlab® function *fit*, [22]. (Table 7) provides the
 568 coefficients of the non-linear gradient estimator and the fitting errors or residuals. Newly,
 569 residuals are compliant with target radiosity uncertainty, and standard errors for the coefficients
 570 have not been provided because of the small number of data (20 points) that leads to high and
 571 not real uncertainties. The non-zero and very small value for the independent terms, K_{sp0} , it is
 572 associated with an uncertainty in the temperature or irradiance of the calibration target, that
 573 leads to identify an unreal gradient in the thermopile package.

Channel	K_{sp0} [mK]	K_{sp1} [mK/W]	K_{sp2} [mK/K]	K_{sp3} [mK/W ²]	K_{sp4} [mK/W·K]	Fit residuals-1 σ [mK]
IR1	1.83	166.5	9.63	-95.1	1.85	4.1
IR2	0.60	1.3	1.03	-7.2	-0.80	0.27
IR3	0.42	166.4	10.40	12.5	15.47	2.7
IR4	1.20	114.6	11.69	-105.6	-3.94	3.9
IR5	-0.20	55.6	6.82	-56.1	-2.56	2.0

574 Table 7. Gradient estimators and fit residuals for the calibration mode using the support plate

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576 5.4 Discussion

577 The values of the obtained experimental thermal gradient for the IR3 channel, shown in Fig 10.c,
 578 coincide to a large extent with those predicted by the TMM, since the results of this type of
 579 thermopile for a different model were used in the adjustment of the TMM parameters and thus
 580 they are mathematically forced (section 4). This similarity is revealed mathematically in the
 581 values of K' , $-3.612 \text{ mK/K} \cdot h$ for TMM versus $-3.861 \text{ mK/K} \cdot h$ for experimental of IR3
 582 (Table 5). Nevertheless, for the other coefficient (K) of channel IR3 and for the rest of channels
 583 coefficients, experimental gradients differ significantly from the TMM results. Several factors
 584 justify these results:

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586 i) First, the thermopile's filter covers a large part of the thermopile internal view factor (Fig.
 587 4b) and factors such as filter optical properties (emissivity and reflectance) and the
 588 reflections inside the package are aspects that were not included in the thermopile's model,
 589 but that may affect the radiative exchange inside the package. Thus, the lack of filter
 590 modeling and the use of Stefan Boltzmann's law represent a clear simplification.

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592 ii) Second, the differences in the thermal properties of the filter substrate, such us thermal
 593 conductivity, heat capacity and density and IR emissivity, modify the value and the temporal
 594 evolution of the gradients. (Table 8) shows the value of some of these quantities for the
 595 different substrate of the filters [23]. The thermal time constant of the filter, $\tau \sim c_p \cdot \rho / \lambda$, is
 596 directly proportional to the specific heat (c_p) and density (ρ) and inversely proportional to
 597 the thermal conductivity (λ). This implies that the quartz filter presents a time constant 8.5
 598 times larger than germanium, and 22 times larger than silicon. This is consistent with the
 599 temporal responses shown in (Fig. 11b insert). On the other hand, the steady state thermal
 600 gradients between the filter (package front side) and the thermopile's package are inversely
 601 proportional to filter emissivity (ε) and the filter thermal conductivity (ρ). This dependence
 602 is clearly observed when we compare the value of the gradients between IR2 and IR3. Both
 603 filters present similar emissivity but a significantly different thermal conductivity, which
 604 implies a greater gradient in the case of IR3 given the lower thermal conductivity of the
 605 quartz. Similarly, IR5 and IR4 channels share Si as the filter substrate, and it is the lower
 606 emissivity of the IR4 channel filter that causes the greater gradient for this channel.

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iii) Finally, another important aspect is related to the constructive details, both internal to each thermopile sample and external, such as its orientation and fixation to the support plate. One example is the amount of glue used and its distribution, both in the bonding of the filter and in the bonding of the thermopile to the support plate. This could explain the differences in the gradients for the case of channels with the same filter substrate but different orientation, as in the case of channels IR1 and IR4, as the TMM results do not justify any difference due upward or downward facing orientation (section 4.1).

Channel	Filter substrate	Emissivity (ϵ)	Thermal conductivity (λ) ^a ~@300K [W/K·m]	Specific heat (c_p) ^a ~@300K [J/K·Kg]	Density (ρ) ^a [Kg/m ³]
IR1 & IR4	Si	~0.53 ^b	148	714	2330
IR5	Si	~0.7 ^b	148	714	2330
IR2	Ge	~0.97 ^b	59.9	320	5323
IR3	Quartz crystal	~0.93	12-6.8 ^c	733	2650

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^a Assuming pure substances

^b Based on filter transmittance (τ) data and assuming $\epsilon=1-\tau$ for the filter high emissivity side

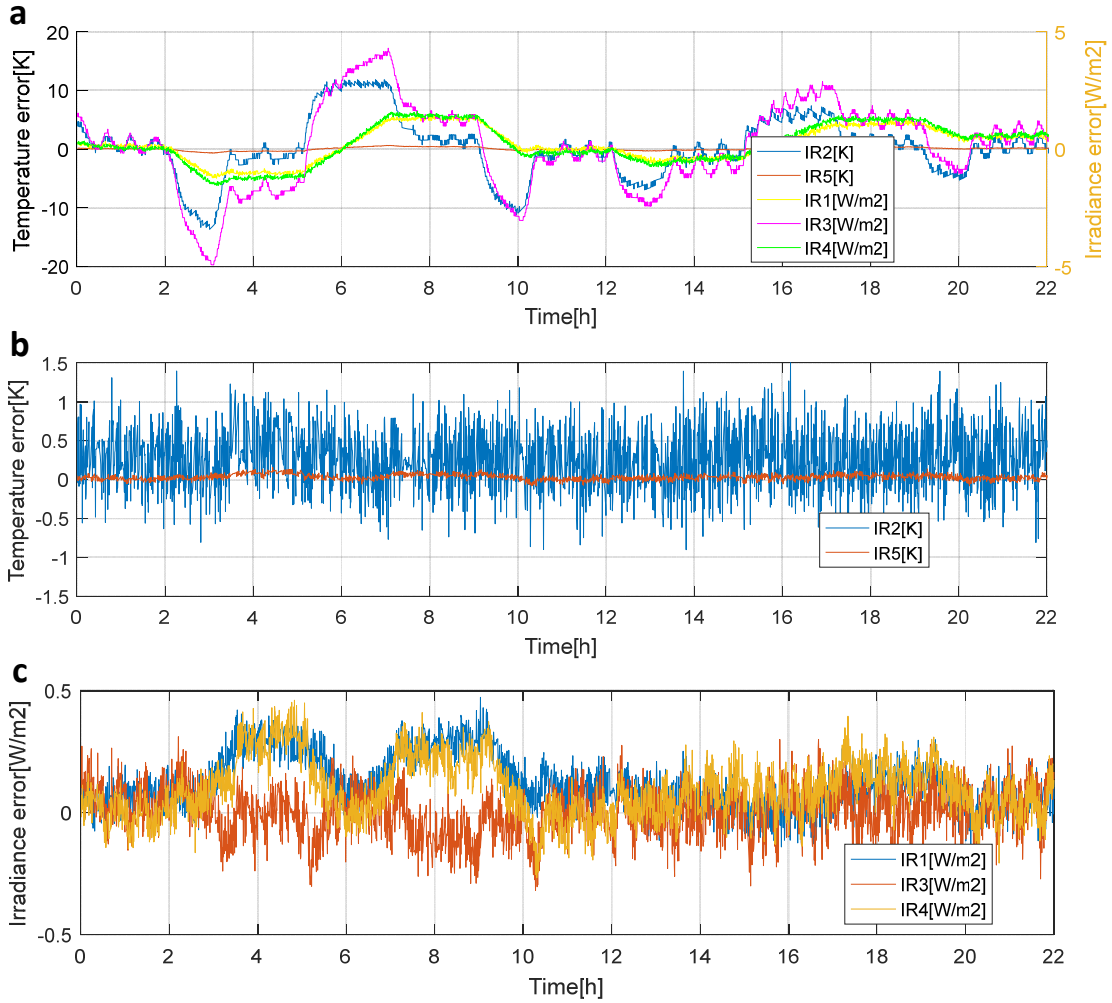
^c Depending on the direction parallel or perpendicular to crystal c axis

Table 8. Thermal and optical properties of filter substrate

6. Thermal calibration performance and uncertainties

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The results of the measurement errors for the different TIRS channels, before and after thermal gradients compensation for the calibration data set of the nominal mode, are presented in (Fig. 14). The errors are mainly associated to the thermopile's package thermal gradients and they are larger for channels IR2 and IR3 (Fig. 14a) because of their narrower bandpass filters and therefore their smaller signal-to-package's gradient ratio [15]. (Fig. 14b and 14c) show the excellent performance of the thermal estimators, reducing significantly the errors for channels IR2 and IR3 in a factor between 10 and 20, respectively. For channels IR1, IR4 and IR5 the improvement, although existing, is not so clear, being conditioned by errors in the radiosity of the calibration target. In addition to that, the compensation of the thermal gradients increases the associated noise because of the use of a derivative in the compensation (Eq. 3.1). The support plate temperature derivative has been computed using finite differences, in which a 6 samples moving average filter has been used to minimize the associated noise (sampling time 30 sec during calibration test). This allows a noise reduction of up to 0.33K (1σ) for IR2.



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Figure 14. Target temperature or irradiance error for the different channels due to the thermopile package thermal gradients for the nominal operation calibration data set, (a) before applying thermal gradient compensation, and (b) temperature error after thermal gradient compensation, (c) irradiance error after thermal gradient compensation

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The compensation of the thermopiles' package thermal gradients during nominal operations is affected by uncertainties, which originate from the following error sources:

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(i) The limitations of the test setup, with respect to the inability to cover the entire operational temperature range and solar loads [13], affects the validity of the gradient estimation functions during Martian operations. To calculate the associated uncertainty, the TMM results has been used to extend the validity of the tests to different thermal environments. The TMM results for channel IR3 provide uncertainties for the estimation of K_{TMM} and K'_{TMM} , $\Delta K_{TMM} = \sigma_{K_{TMM}}/K_{TMM} = 4.61\%$ and $\Delta K'_{TMM} = \sigma_{K'_{TMM}}/K'_{TMM} = 0.14\%$ respectively. These two errors are propagated, taking partial derivatives in (Eq. 3.1) and adding quadratically the uncertainties. Thus, the uncertainty can be calculated according to (Eq. 6), where the values of $|T_{cp} - T_{sp}|_{max} = 5.6K$ and $|\partial T_{sp}/\partial t|_{max} = 20K/h$ are based on the results of the TMM, and K and K' are the coefficients for each channel result of the estimation form the calibration test (Table 5). The results are shown in (Table 9).

654

$$\Delta(T_{sf} - T_s) = \sqrt{\left(K \cdot \Delta K_{TMM} \cdot |T_{cp} - T_{sp}|_{max}\right)^2 + \left(K' \cdot \Delta K'_{TMM} \cdot |\partial T_{sp}/\partial t|_{max}\right)^2} \quad (6)$$

655

656

(ii) The uncertainty in the determination of calibration target radioisity, φ_t , equivalent to a calibration target temperature error of ± 0.3 K. From the mathematical model of the

657 detectors (Eq. 1.1 and 1.2) an equivalent thermal gradient error can be computed using the
 658 real temperatures for target and TIRS during the test (Table 9). This error is very large for the
 659 channels IR1, IR4 and IR5, which should be reduced in future work.

660 (iii) The practical fit uncertainty of the gradient estimation functions. The procedure to
 661 calculate this uncertainty makes use of the standard errors of K and K' during its calibration
 662 (Table 5). The errors are propagated, taking partial derivative in Eq. 3.1 and using the
 663 maximum values for $|T_{cp} - T_{sp}|_{max}$ and $|\partial T_{sp}/\partial t|_{max}$ taken from the TMM. (Eq. 7) summarizes
 664 the procedure, and results for each channel are shown in (Table 9).

$$665 \Delta(T_{sf} - T_s) = \sqrt{(\sigma_K \cdot |T_{cp} - T_{sp}|_{max})^2 + (\sigma_{K'} \cdot |\partial T_{sp}/\partial t|_{max})^2} \quad (7)$$

Channel	Testing temperature limitations (σ) [mK]	Calibration target equivalent temperature error (σ) [mK]	Gradients estimators (σ) [mK]
IR1	3.54	9.13	0.17
IR2	0.50	0.358	0.06
IR3	3.85	0	0.17
IR4	4.19	9.27	0.20
IR5	2.38	5.73	0.11

666 Table 9. Thermal calibration uncertainties expressed in terms of equivalent thermopiles'
 667 package thermal gradients errors.
 668

669 Finally, Table 10 summarizes the total calibration uncertainty for the nominal mode of
 670 operation, and the equivalent target temperature or irradiance uncertainty. They have been
 671 calculated for extreme operational temperatures and a nominal thermal environment [6], where
 672 ground temperature is equal to $T_s - 20K$, sky temperature is $T_s - 70K$ and atmosphere temperature
 673 is $T_s - 10K$. To compute these figures the quadratic addition of the different uncertainties sources
 674 was used, assuming uncorrelated probability distributions for all of them.
 675

Channel	Total uncertainty (1σ) [mK]	Nominal mode (Equivalent uncertainty -1σ $T_s=298K$)	Nominal mode (Equivalent uncertainty -1σ $T_s\sim 173K$)
IR1	9.79	2.08W/m ^{2a}	1.93W/m ^{2a}
IR2	0.62	0.55K	0.54K
IR3	3.85	0.49W/m ²	0.49W/m ²
IR4	10.17	1.98W/m ^{2a}	1.98W/m ^{2a}
IR5	6.21	0.40K	0.34K

676 ^a This error is calculated assuming a blackbody model for the target and Stefan-Boltzmann emission
 677 Table 10. Thermal gradient total uncertainty and equivalent target temperature or irradiance
 678 uncertainty for two different detector temperatures
 679

680 In-flight calibration is a differential process aimed at eliminating constant uncertainties by
 681 comparing measurements before and during the activation of TIRS heaters [6, 17]. Under this
 682 premise, a basic and preliminary analysis of the uncertainty associated with the calibration of
 683 the thermal gradients during the in-flight calibration modes can be carried out. In this analysis,
 684 it has been assumed that the fit residuals of the practical tests (Tables 6 and 7) correspond to
 685 the uncertainty value in the compensation of the thermal gradients, as no errors were identified
 686 for each of the estimator constants. By using the sensor model (Eq. 1), the fit residuals of the
 687 thermal gradients can be transformed into an equivalent error in the temperature of the
 688 calibration or support plates, depending on the calibration mode (Table 11).

689 Since the expected heating of the calibration plate during the in-flight calibration is around 15K
 690 (Fig. 11a), the equivalent temperature errors associated with the uncertainty in the gradient
 691 compensation (Table 11) take values between 1.1% and 3.5% of the temperature increment.

692 Similarly, a temperature increment of around 7K for the heating of the support plate is expected
 693 (Fig. 11a), with resulting associated errors between 0.9% and 1.6%. These uncertainties are
 694 compatible with the expected performance of the in-flight calibration system in the reevaluation
 695 of the detector responsivity, established at 3% in (Table 3) of reference [6]. As future work, a
 696 more exhaustive analysis of the performance of the flight calibration modes will be performed
 697 by means of an end to end test, making use of the gradient estimators obtained in this article.
 698

Channel	Calibration plate equivalent temperature error (σ) [K]	Support plate equivalent temperature error (σ) [K]
IR1	0.22	0.08
IR2	0.52	0.11
IR3 ^a	-	-
IR4	0.20	0.08
IR5	0.16	0.06

^a Channel IR3 cannot be recalibrated using in-flight calibrations [6, 12]

699
 700 Table 11. Equivalent calibration or support plate temperature error for the calibration modes
 701

702 7. Conclusions

703 The thermal gradients in the package of the IR detectors of Mars 2020 MEDA/TIRS instrument
 704 affects its accuracy, making it necessary to compensate for their effects. This is particularly the
 705 case for detectors with reduced spectral bandwidth such as channels IR2 and IR3. The thermal
 706 calibration approach makes use of a simplified mathematical model [12] that accounts for such
 707 thermal gradients. Moreover, a set of time dependence and steady state equations for gradient
 708 estimation and compensation was defined, considering the different operational modes of TIRS.

709 TIRS TMM results have been used in the definition of the estimators, showing a high degree of
 710 correlation between the thermopiles' package spatial gradients and: (i) the temperature change
 711 rate experienced by the support plate where they are allocated, (ii) the spatial gradient between
 712 the support and calibration plate, and (iii) the power driven to the calibration heaters.

713 Calibration tests were performed to identify the coefficients of the functions for the estimation
 714 of the gradients. The results of the tests show important differences in the values of the
 715 gradients for the different types of thermopiles. This highlights, on the one hand, the
 716 simplification imposed by the mathematical modeling, and on the other, the effect of the IR
 717 filter material and its mounting in the value of the gradients.

718 The factors affecting the thermal calibration accuracy for the nominal operational mode were
 719 identified and assessed. These factors are: (i) The standard errors in the coefficients of the
 720 gradients estimation functions, (ii) the calibration target temperature accuracy, and (iii) the
 721 simplifications that the calibration setup imposes regarding operational temperature. For this
 722 last factor, the TMM results were used to estimate an upper bound for the error caused by the
 723 use of our experimental setup, since the test was performed at a unique temperature.

724 The thermal calibration resulted in a significant improvement in the TIRS performance and
 725 accuracy for channels IR2 and IR3 in the nominal mode, versus an approach in which thermal
 726 gradients are not compensated, and it allows carry out the in-flight calibration with the expected
 727 performance. The total uncertainty associated with the thermal calibration of the different
 728 channels range from 0.62 to 10.17mK, improving the 22 mK obtained in [6] as a preliminary
 729 value for all the TIRS channels. These numbers can be improved using a better calibration setup
 730 that reduce the uncertainty in the radiosity of the calibration target [10]. However, the current
 731 uncertainties meet the scientific requirements of the radiometer established by the Mars 2020
 732 science team (<1% of irradiance and 5K in temperature) [6], validating the thermal calibration
 733 approach.

734

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